

Modeling solar desalination with reverse osmosis (RO) powered by concentrating solar power (CSP) plant

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Abstract – This article deals with the desalination of seawater and brackish water, which can deal with the problem of water scarcity that threatens certain countries in the world; it is now possible to meet the demand for drinking water. Currently, among the various desalination processes, the reverse osmosis technique is the most used. Electrical energy consumption is the most attractive factor in the cost of operating seawater by reverse osmosis in desalination plants. Desalination of water by solar energy can be considered as a very important drinking water alternative. For determining the electrical energy consumption of a single reverse osmosis module, we used the System Advisor Model (SAM) to determine the technical characteristics and costs of a parabolic cylindrical installation and Reverse Osmosis System Analysis (ROSA) to obtain the electrical power of a single reverse osmosis module. The electrical power of a single module is 4101 KW; this is consistent with the manufacturer's data that this power must be between 3900 kW and 4300 KW. Thus, the energy consumption of the system is 4.92 KWh/m³. Thermal power produced by the solar cylindrical-parabolic field during the month of May has the maximum that is 208MWh, and the minimum value during the month of April, which equals 6 MWh. Electrical power produced by the plant varied between 47MWe, and 23.8MWe. The maximum energy was generated during the month of July (1900 MWh) with the maximum energy stored (118 MWh).

Keywords: System Advisor Model (SAM), Reverse Osmosis System Analysis (ROSA), Rankine cycle, parabolic trough power.

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I. Introduction

The use of seawater desalination to provide fresh drinking water is a well-established and flourishing industry. The two main technologies used are thermal desalination and Reverse Osmosis (RO) membrane filtration. The desalination of brackish water or seawater by conventional treatment units requires a large electrical energy and / or heat. Among the processes, distillation and reverse osmosis are technologies whose performance has been proven to desalinate salt water [1]. Concentrating solar power (CSP) technology is considered as one of the alternative solutions of power generation from solar energy. In this technology, sunrays are focused onto a solar receiver with the help of mirrors. The energy captured by the receiver is converted to heat or electricity through a series of process [2]. Martin et al. used a mathematical programming technique to optimize the operation of a CSP plant employing regenerative Rankine cycle for a site in Spain. [3]. A simulation model for predicting the electrical output of a 50 MWe PTC power plant was developed by Garcia et

al. [4]. The model results were compared with the experimental data of a power plant operating in Spain.

During our modeling and simulation work, we used software, SAM [5] (System Advisor Model) for the energy study of a parabolic concentrator and ROSA [6] (Reverse Osmosis System Analysis) to obtain the electrical power of a single reverse osmosis module.

The objective of this work is the technical and economic study of a parabolic plant used to produce electricity consumed by a reverse osmosis module. The results obtained make it possible to highlight the effect of the electrical energy consumption on the cost of desalination by reverse osmosis.

II. Presentation of the solar desalination plant used

The heat generated by a CSP plant can be utilized to produce the required electric power needed to drive the

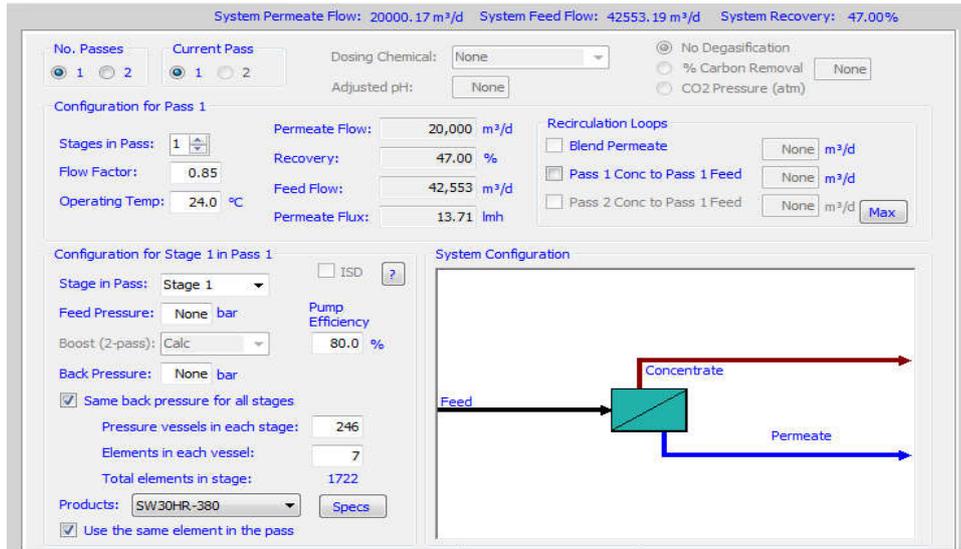


Figure 2. The configuration of the reverse osmosis system on ROSA

III.2. Rankine cycle with solar field

After having completed the first step concerning the reverse osmosis module and the definition of the power necessary to operate the reverse osmosis desalination unit, we will proceed to the next step, which concerns the dimensioning of the CSP plant in order to provide a power supply for the reverse osmosis unit. We used another technical-economic software to determine the technical characteristics and costs of the CSP system, as well as the performance analysis of the parabolic cylindrical CSP plant.

In this part, we will make a simulation of a CSP plant, whose heat transfer fluid is thermal oil (Therminol VP1), this plant is tested for the Ain Témouchent site, the nominal power of this plant is 40.7 MWe. The System Advisor Model (SAM) code was used to simulate the

annual behavior of the CSP plant at the time scale. SAM predicts the dynamic behavior of the parabolic trough.

The System Advisor Model (SAM) software is a software that performs cost and performance analysis of solar installations. This software was developed by the National Renewable Energy Laboratory (NREL), Sandia's national laboratories in partnership with the United States Department of Energy (DOE), and the Solar Energy Technology and Program (SETP). It has been designed to facilitate decision-making for those involved in the renewable energy industry [5].

The empirical model of a CSP modeling plant is identical to the physical model, only it uses a set of curve fitting equations derived from the regression analysis of the measured data from SEGS (Solar Energy Generating Systems) projects in the southwestern United States, is shown in Figure 3.

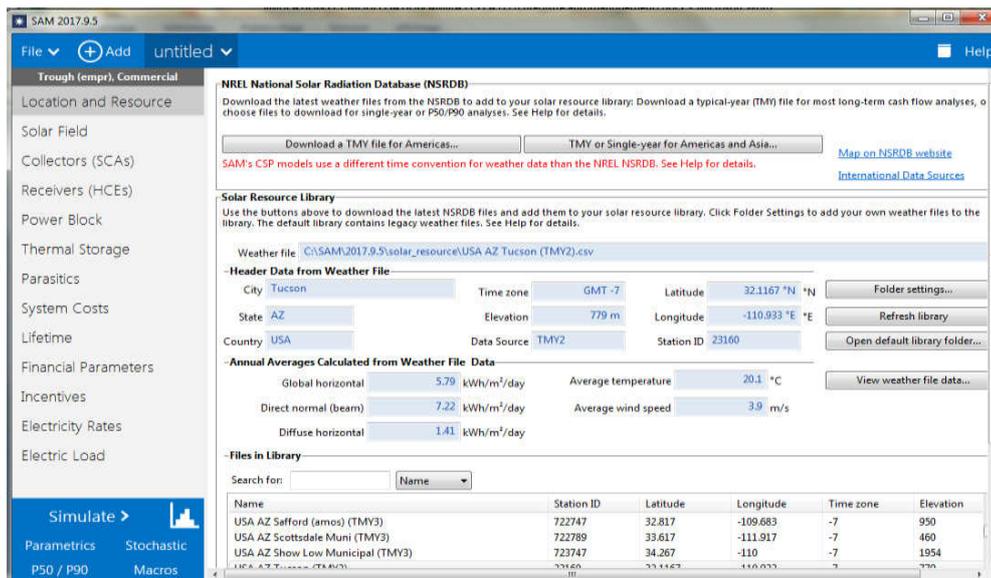


Figure 3. The CSP plant in SAM program

The parameters concentrating the different components of the installation are:

- 1-Solar Field
- 2-Collectors (SCAs)
- 3-Receivers (HCEs)
- 4-Power Block
- 5-Thermal Storage

Figure 4a Shows the solar radiation, the most extreme estimations of solar radiation was 1000 W.m^{-2} . Figure 4.b shows the change in ambient temperature and Figure 4c shows the wind speed during the year for the selected site. Note that the maximum and minimum temperature recorded is equal to 37° C and 1° C respectively and we note that the maximum wind speed is 15 m/s .

IV. Results and discussion

IV.1. Meteorological data

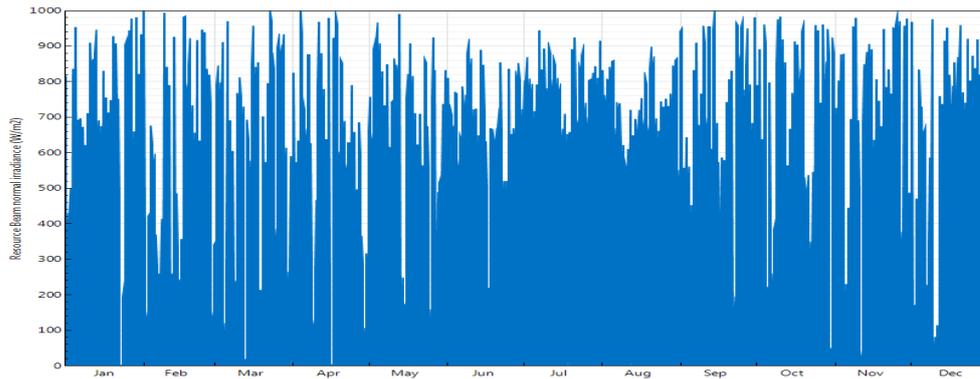


Figure 4a. Shows the solar radiation

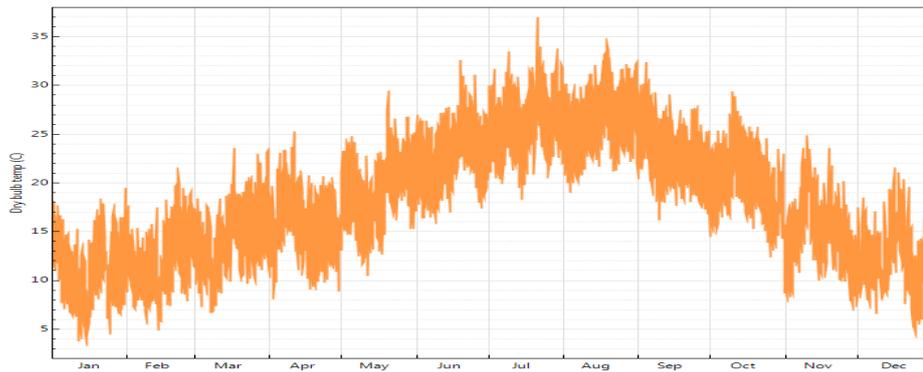


Figure 4b. Variation of ambient temperature

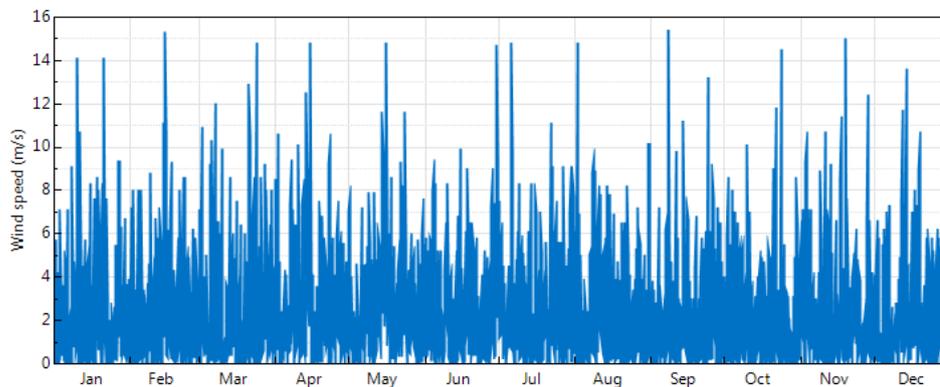


Figure 4c. Variation of wind speed

IV.3. System performance of CSP plan

IV.2. Reverse osmosis

After introducing, the previous data into the ROSA code. The simulation results shown in Table 2, we note that the electrical power of a single module is 4101 KW; this is consistent with the manufacturer's data that this power must be between 3900 kW and 4300 KW. Thus, the energy consumption of the system is 4.92 kWh/m³.

Table2: RO system details

Parameter	Value
Feed flow rate [m ³ /j]	42553.19
pressure brine [bar]	66.61
brine flow rate [m ³ /j]	22553.02
pressure Permeate [bar]	65.20
Permeate flow rate [m ³ /j]	20000.17
salinity [mg/l]	277.51
The power [kw]	4101.57
Specific energy [kwh/m ³]	4.92

IV.3.1. Thermal power produced by the solar cylindro-parabolic field

Figure 5 shows the total heat output produced during the year. We note that the thermal power during the month of May has the maximum that is 208MWth, and the minimum value of the thermal power recorded during the month of April, which equals 6 MWth.

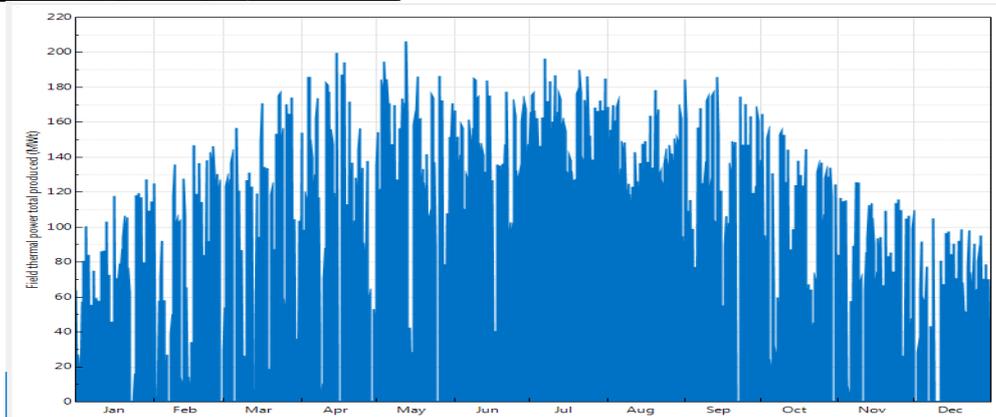


Figure 5. Total thermal power produced during the year (MWt)

IV.3.2. Electrical power produced by the plant

Figure 6 shows the hourly variation in the net power generated during the year. Note that the power is varied

between 47MWe, and 23.8MWe. It is observable that the power produced is proportional to the incident direct radiation on the solar reflectors.

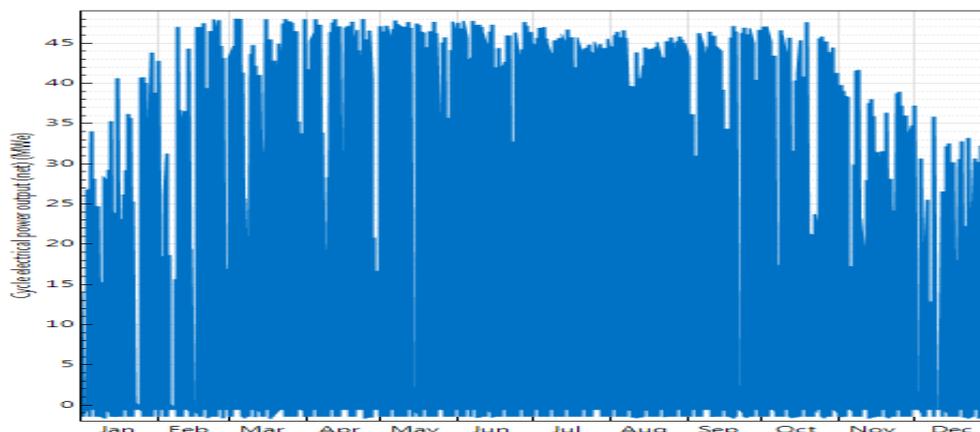


Figure 6. System power generated during the year (KWe)

IV.3.3. Power of CSP plant

As can be seen in the graphs above that: The maximum thermal power absorbed was estimated at 12 Mwt as well as the maximum electrical output power (net) was estimated at 4.5 Mwe, and the maximum electrical power of production (gross) was estimated at 4.8 Mwe and Incident of maximum thermal power on the ground was estimated at 30 Mwt .its maximum thermal loss of the reservoir was estimated at 0 Mwt and TES Maximum thermal energy in storage was estimated at 4 Mwt.

IV.3.4. Energy generated and thermal energy in storage

The energy generated by each month and the thermal energy for storage illustrated in Figure 7. It can be seen that the maximum energy was generated during the

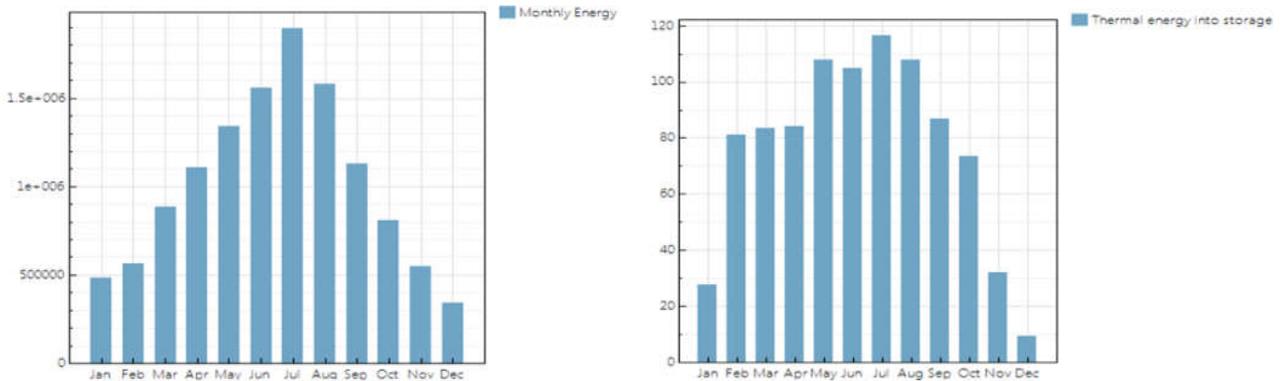


Figure 7. Monthly energy generated and thermal energy in storage (MWh)

month of July (1900 MWh) with the maximum energy stored (118 MWh).

IV.3.5. Field HTF temperature hot and cold header

The heat transfer fluid (HTF) flows from the cold reservoir to the hot reservoir, the minimum temperature of the cold header inlet has reached a value of 160 °C, and the maximum temperature obtained at the header outlet hot is 400 0C. The variation of the temperature of the hot and cold header input is shown in the Figure 8.

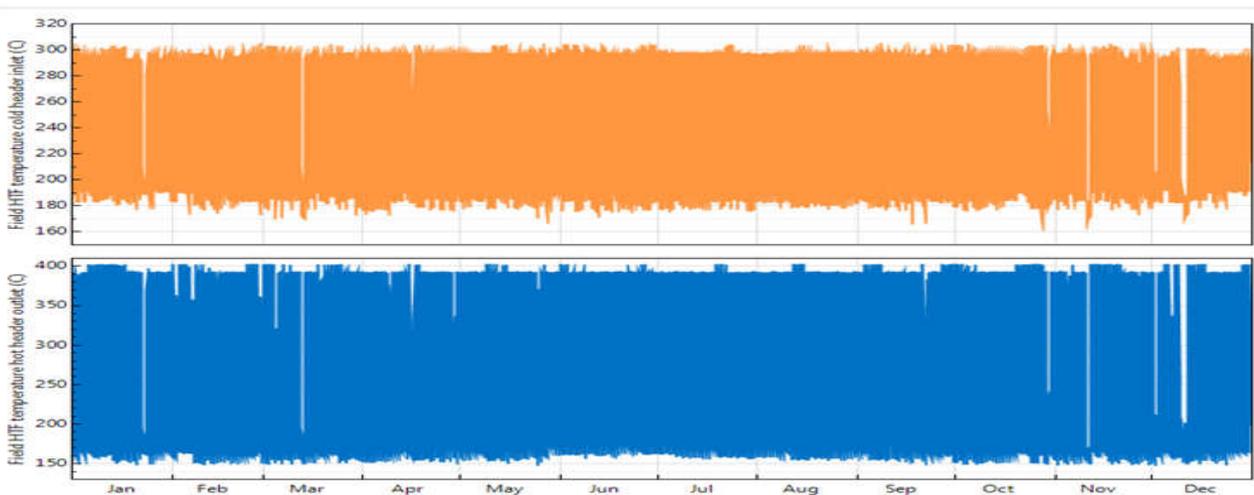


Figure 8. Field HTF temperature hot and cold header

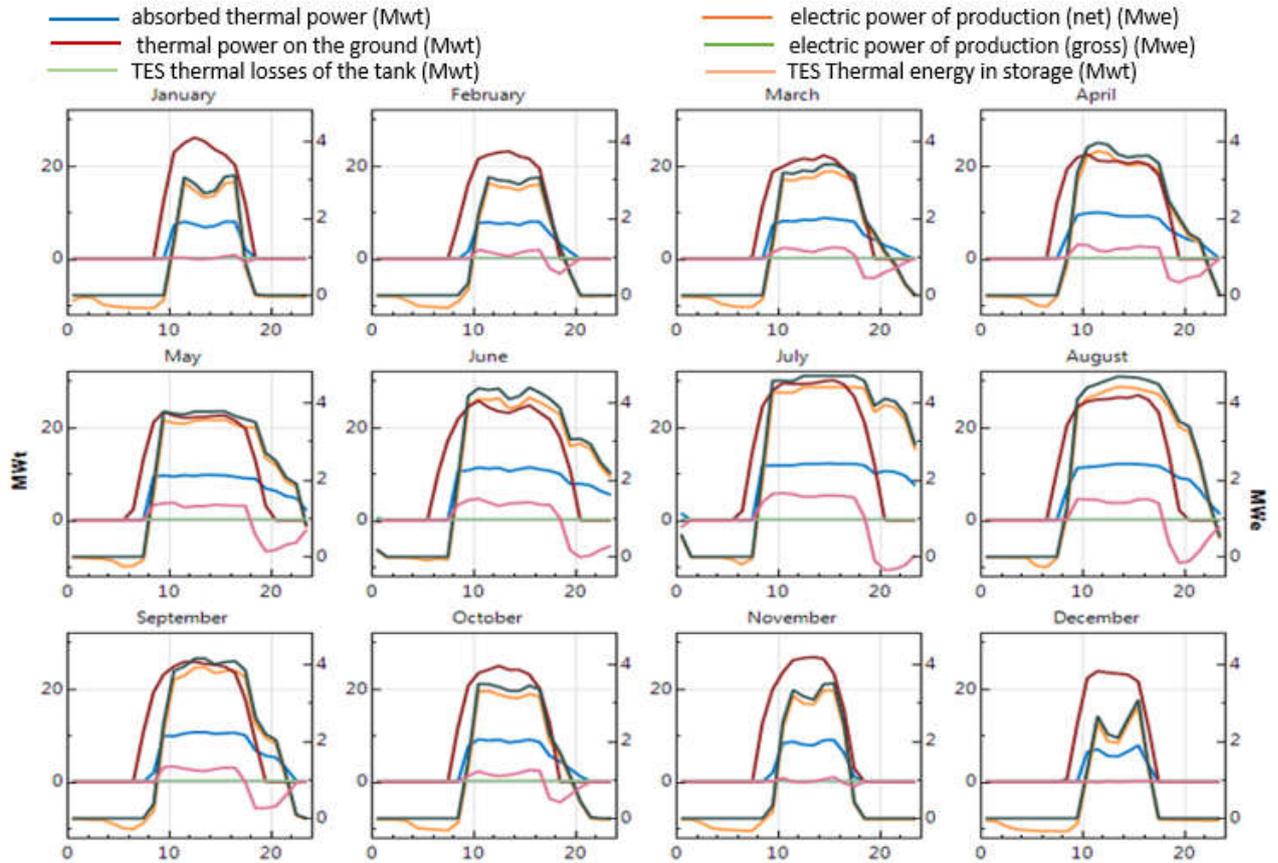


Figure 9. Absorbed thermal power (Mwt) and field power (Mwt), gross and net electric power (Mwe), TES thermal losses from the tank (Mwt) and in storage (Mwt)

volume is 1000 m^3 and the maximum cold tank HTF volume is 1200 m^3 as shown in Figure 9 and Figure 10.

IV.3.6. TES HTF, hot and cold tanks volume

For power generation after sunset, the HTF fluid should be stored in the storage tank for a storage capacity of 8:00 h. The total HTF maximum thermal energy storage (TES) is 1300 m^3 , the maximum hot tank HTF

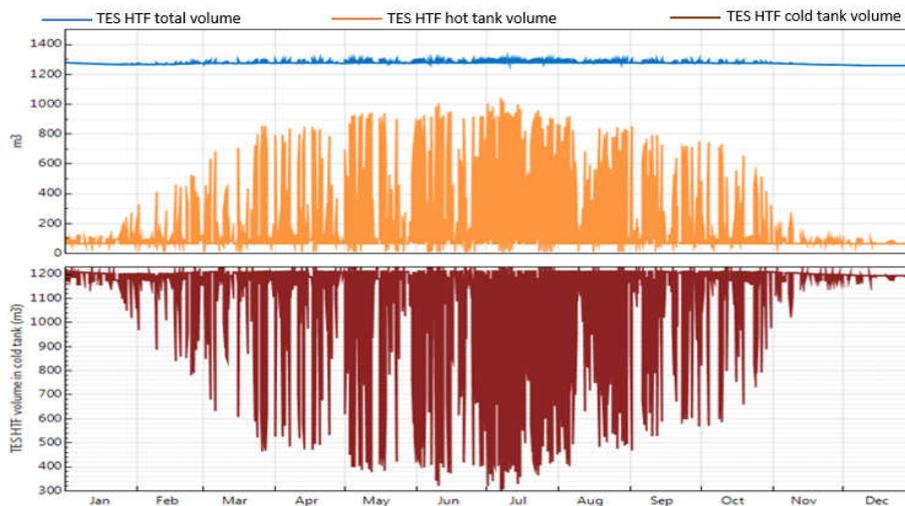


Figure 10. TES HTF total volume, hot and cold tanks volume

IV.4. Technical characteristics and system costs

The simulation is carried out using the SAM software, the technical characteristics of the CSP system are shown in Table 3 and the system costs are shown in Table 4.

Table 3: Technical characteristics of CSP system

	The variables	Values
Collector	Reflective Opening Area [m ²]	817.5
	Opening width, total structure [m]	5.75
	Length of collector assembly [m]	150
	Number of modules per assembly [-]	12
	Average length of the surface at the focal length [m]	2.11
	Distance between assemblies [m]	1
Receiver	Inside diameter of absorber tube [m]	0.076
	Outer diameter of absorber tube [m]	0.08
The solar field	Line spacing [m]	15
	Irradiation [W/m ²]	950
	Thermal field production [MW]	23.5613
	Solar Field Area (acres)	25
	Total Area (acres)	35

Table 4. CSP system costs.

The costs	Values
Site Improvements (\$)	981000
Solar field (\$)	5 886 000
HTF system (\$)	2354400
Storage (\$)	6 125950
Total direct cost (\$)	22 120782
Total Indirect Cost (\$)	3 672293
Total cost of installation (\$)	25 793022
Total cost of the facility per capacity (\$ / kW)	6 833

V. Conclusion

This article focuses on the techno-economic modeling of a cylindro-parabolic power plant to generate the electrical energy required for desalination by reverse osmosis. The results show that:

- The electrical power of a single module is 4101 KW; this is consistent with the manufacturer's data that this power must be between 3900 kW and 4300 KW. Thus, the energy consumption of the system is 4.92 KWh/m³.
- Thermal power produced by the solar cylindro-parabolic field during the month of May has the maximum that is 208 MWth and the minimum value during the month of April, which equals 6 MWth.
- Electrical power produced by the plant varied between 47MWe, and 23.8MWe
- The maximum energy was generated during the month of July (1900 MWh) with the maximum energy stored (118 MWh).
- The minimum temperature of the cold header inlet has reached a value of 160 °C, and the maximum temperature obtained at the header outlet hot is 400 °C.
- Total cost of the facility per capacity is 6.833 \$/kW

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